



Technical Report 1

*Evaluation of ASHRAE 62.1-2010: Ventilation for Acceptable Indoor Air Quality
& ASHRAE 90.1-2010: Energy Standard for Buildings except Low Rise
Residential*

September 18, 2013

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Executive Summary

Technical Report 1 evaluated the compliance of the Auditorium, a performing arts theater and office space, located in Lemma, MN. The building is owned and operated by Francis Michael Performing Arts Academy (FMPAA) and was originally constructed in 1929. The Auditorium was recently renovated to accommodate the needs of FMPAA and is set to be completed in April 2014. The basis of this evaluation is ASHRAE Standard 62.1-2010 Ventilation for Acceptable Indoor Air Quality and ASHRAE Standard 90.1-2010 Energy Standard for Buildings except Low Rise Residential.

Through investigation of ASHRAE 62.1-2010, the Auditorium falls in accordance with the requirements set forth by the standard to protect the health of the building occupants and exceeds the requirements in some areas. The systems and controls in place are designed to prevent the growth of mold, provide adequate ventilation, and minimize the amount of contamination through cleaning and maintenance.

After assessing the Auditorium in relation the requirements for ASHRAE 90.1-2010, the building falls under a general compliance. All of the heating, cooling and ventilation equipment are acceptable and even exceed the standard in some areas, however the building envelope falls short of the thermal requirements. Considering that this building was originally built in the early 1900s, adding more insulation or increasing the tightness of construction could cause more harm to the building in terms of moisture build-up.

Building Overview

The Auditorium is a historic building located on the campus of the Francis Michael Performing Arts Academy (FMPAA). It was built in 1929, and has recently undergone a renovation to revitalize the performance space and allow for greater usage of the ancillary public spaces. After completion of construction the Academy Honors Program will permanently reside in the Auditorium

A pediment entrance way with ionic columns faces the prominent campus mall. The building facade is a 3 wyth historic brick construction with classical ornamentation. The building is approximately 172,000SF, five stories tall and located in the very cold climate of Lemma, Minnesota.

The plan below (Figure 1) shows the expanded performance space (green), audience chamber (maroon), and horseshoe of public office spaces (orange) surrounding.

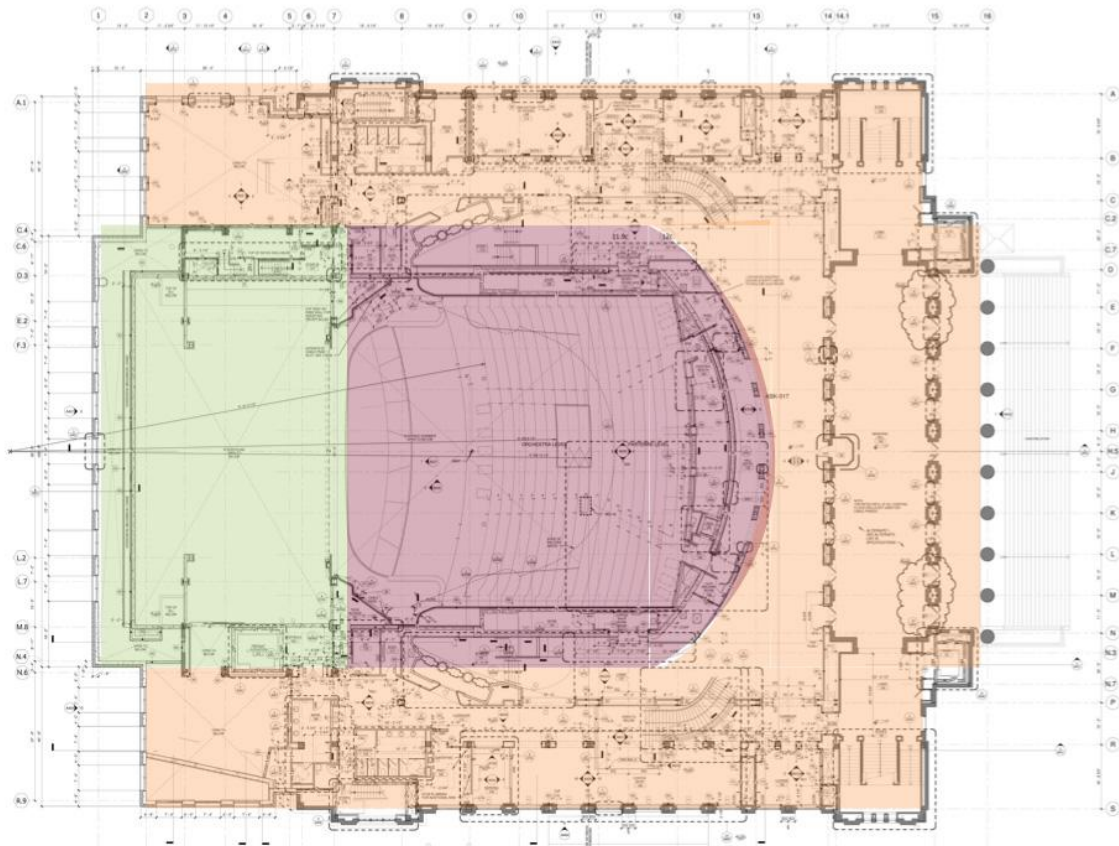


Figure 1 First Floor Level (Source: Architect of Record)

Mechanical Systems Overview

The mechanical system of the auditorium employs several technologies to distribute heating and cooling to the building occupants. The primary heat source for the building is from a campus steam plant. The steam plant provides 150°F steam to a flooded high pressure heat exchanger to create hot water. The hot water is then distributed to fin tube radiation units, fan-powered boxes and four air handling units. Steam is also utilized in the air handling units humidification systems.

Located in the basement of the Auditorium is the campus cooling plant. It includes three -1000 ton centrifugal chillers, which accommodate the northwest corner of campus including the Auditorium. Chilled water is distributed to the air handling units, in addition to the active chilled beams which serve the performance support spaces.

Four air handling units serve the building. Each unit is sized to accommodate the following program spaces:

- AHU-1: Public Spaces - Variable Air Volume
- AHU-2: Audience Chamber - Displacement Ventilation via Underfloor Air Distribution
- AHU-3: Performance Spaces - Variable Air Volume
- AHU-5: Performance Support Spaces - DOAS with dual-energy recovery wheel
- *Note: AHU-4 was not used and does not exist in the final construction documentation*

AHU-5 is a dedicated outdoor air system (DOAS) and has a dual-energy recovery wheel that serves the active chilled beam system. The design team approximates that it reduces the amount of conditioned air by 35%. Please see the end of section 6.2 Ventilation Rate Procedure for the breakdown and calculations of airflow values for each of air handling units.

ASHRAE Standard 62.1 - 2010 Evaluation

This section will analyze how the mechanical design of the Auditorium complies with ASHRAE Standard 62.1-2010. Please note sections 1 - 4 of ASHRAE 62.1-2010 are related to the scope the standard applies to and definitions of terminology used in the standard.

ASHRAE 62.1 Section 5: Systems & Equipment

5.1 Ventilation Air Distribution

The Auditorium is designed to comply with Section 5.1. The drawings indicate appropriate balancing information for compliant ventilation distribution in both ducted and plenum distribution. The drawings indicate requirements for sealing the underfloor supply plenum and reference SMACNA Class A Leakage seal requirements, along with additional leakage testing information.

5.2 Exhaust Duct Location

The exhaust duct locations in The Auditorium are negatively pressurized to mitigate the risk of harmful contaminants leaking into adjacent spaces or air supply or return ductwork. Two general exhaust fans are located on the east (EF-1) and west (EF-2) rooftops rated at an airflow of 4160 cfm and 4350 cfm respectively, to expel contaminants.

5.3 Ventilation System Controls

Francis Michael Performing Arts Academy requires all equipment control systems to interface with their Building Systems Automation Center (BSAC). All air terminal units are under the control of the BSAC and interface with occupancy sensors located in regularly occupied spaces to supply required ventilation.

5.4 Air Stream Surfaces

All air stream surfaces are constructed of sheet metal and connected using metal fasteners to comply with section 5.4.1 Resistance to Mold Growth and section 5.4.2 Resistance to Erosion.

5.5 Outdoor Air Intakes

All outdoor intakes are located on the north and west roofs. Each outdoor air inlet is located at least 10 feet from any exhaust or relief outlet, based on Class 2 exhaust air per Table 5-1 in Figure 2 on the following page. All inlets exceed this minimum distance significantly. Additionally, each inlet is protected with a mesh screen and operable louvers that comply with sections 5.5.2 - 5.5.4, protecting the equipment from rain, snow and birds. Each outdoor air handling unit is also designed with access doors for periodic cleaning and maintenance.

TABLE 5-1 Air Intake Minimum Separation Distance

Object	Minimum Distance, ft (m)
Class 2 air exhaust/relief outlet (Note 1)	10 (3)
Class 3 air exhaust/relief outlet (Note 1)	15 (5)
Class 4 air exhaust/relief outlet (Note 2)	30 (10)
Plumbing vents terminating less than 3 ft (1 m) above the level of the outdoor air intake	10 (3)
Plumbing vents terminating at least 3 ft (1 m) above the level of the outdoor air intake	3 (1)
Vents, chimneys, and flues from combustion appliances and equipment (Note 3)	15 (5)
Garage entry, automobile loading area, or drive-in queue (Note 4)	15 (5)
Truck loading area or dock, bus parking/idling area (Note 4)	25 (7.5)
Driveway, street, or parking place (Note 4)	5 (1.5)
Thoroughfare with high traffic volume	25 (7.5)
Roof, landscaped grade, or other surface directly below intake (Notes 5 and 6)	1 (0.30)
Garbage storage/pick-up area, dumpsters	15 (5)
Cooling tower intake or basin	15 (5)
Cooling tower exhaust	25 (7.5)

Note 1: This requirements applies to the distance from the outdoor air intakes for one ventilation system to the exhaust/relief outlets for any other ventilation system.
Note 2: Minimum distance listed does not apply to laboratory fume hood exhaust air outlets. Separation criteria for fume hood exhaust shall be in compliance with NFPA 45⁵ and ANSI/AIHA Z9.5.⁶ Information on separation criteria for industrial environments can be found in the *ACGIH Industrial Ventilation Manual*⁷ and in the *ASHRAE Handbook—HVAC Applications*.⁸
Note 3: Shorter separation distances shall be permitted when determined in accordance with (a) ANSI Z223.1/NFPA 54⁹ for fuel gas burning appliances and equipment, (b) NFPA 31¹⁰ for oil burning appliances and equipment, or (c) NFPA 211¹¹ for other combustion appliances and equipment.
Note 4: Distance measured to closest place that vehicle exhaust is likely to be located.
Note 5: Shorter separation distance shall be permitted where outdoor surfaces are sloped more than 45 degrees from horizontal or that are less than 1 in. (3 cm) wide.
Note 6: Where snow accumulation is expected, the surface of the snow at the expected average snow depth constitutes the "other surface directly below intake."

Figure 2 (Source: ASHRAE 62.1-2010)

5.6 Local Capture of Contaminants

Equipment generating contaminants is captured at directed outdoors away from any intake openings to be in compliance with section 5.6.

5.7 Combustion Air

The Auditorium does not utilize any combustion equipment therefore this section is not applicable. The heating system for this building is feed from the Academy’s steam plant and converted to hot water through a flooded steam heat exchangers, as stated more thoroughly in the previous section, [Mechanical Systems Overview](#).

5.8 Particulate Matter Removal

All filters specified are at minimum MERV 8 with additional filters rated at a minimum of MERV 11, therefore exceeding the MERV 6 requirement for section 5.8.

5.9 Dehumidification Systems

The outdoor air handling units were selected with a base relative humidity of 50%. This exceeds the 65% requirement. Also, in accordance with section 5.9.2 the total supply airflow far exceeds the exhausted air. Total maximum supply airflow from four outdoor air handling units accounts to 172,000 cfm compared to a total general exhaust airflow of 8,510 cfm.

5.10 Drain Pans

Specification section 233119 on condensate drain pans cites compliance with ASHRAE Standard 62.1-2010. The specification section also indicates that all drain pans shall extend a minimum of 12 inches past the cooling coil.

5.11 Finned-Tube Coils and Heat Exchangers

Drain pans are provided for all finned-tube coils and heat exchangers per section 5.10 to be in compliance with section 5.11. Finned tube coils are installed with 18 inches of access space for cleaning.

5.12 Humidifiers and Water-Spray Systems

The installation of the four types of humidifiers used in the Auditorium; steam injection, electric self-contained, heat exchangers, and electric steam, are specified to be in compliance with ASHRAE 62.1-2010. Each type uses potable water and a drain pan is installed underneath each humidifier.

5.13 Access for Inspection, Cleaning, and Maintenance

Sufficient access to HVAC equipment, ventilation equipment, and air distribution systems has been designed to comply with ASHRAE 62.1-2010 section 5.12. Access doors to clean and inspect air terminal units in ceilings have been included in addition to service spaces in the outdoor handling units. See Figure 3 below for an example schematic of AHU-1 and the access spaces for routine maintenance.

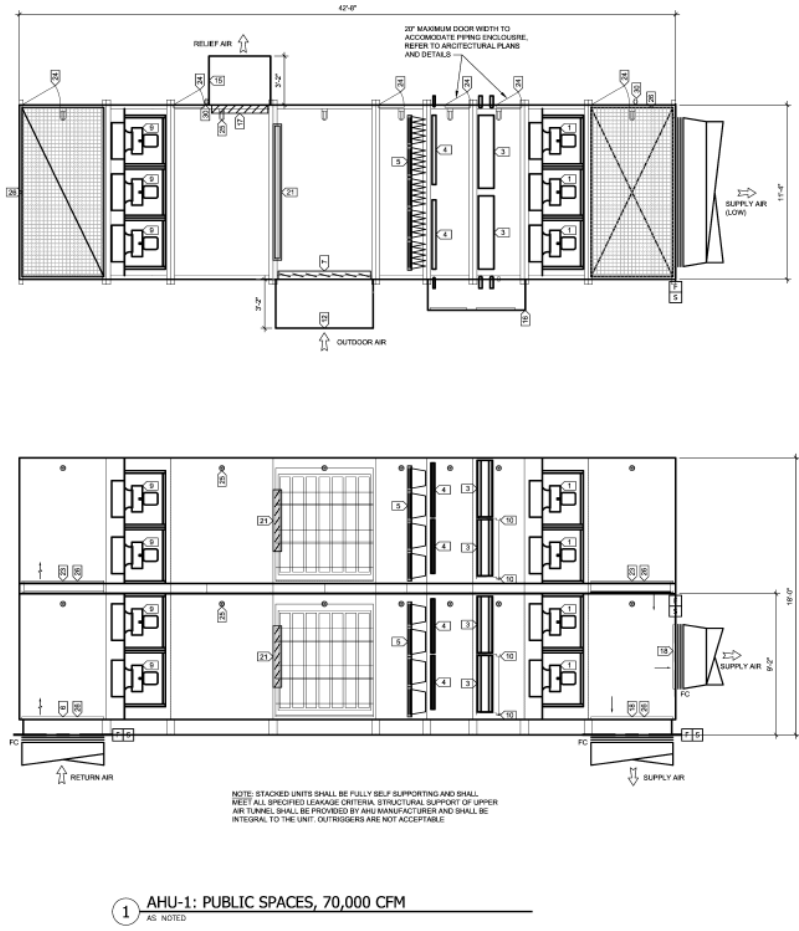


Figure 3 (Source: Architect of Record)

5.14 Building Envelope and Interior Surfaces

The envelope of the Auditorium includes a moisture barrier as indicated in Figure 4 from architectural section detail on sheet A462. Additionally, specification section 230719 details what piping systems are to be insulated to prevent condensation.

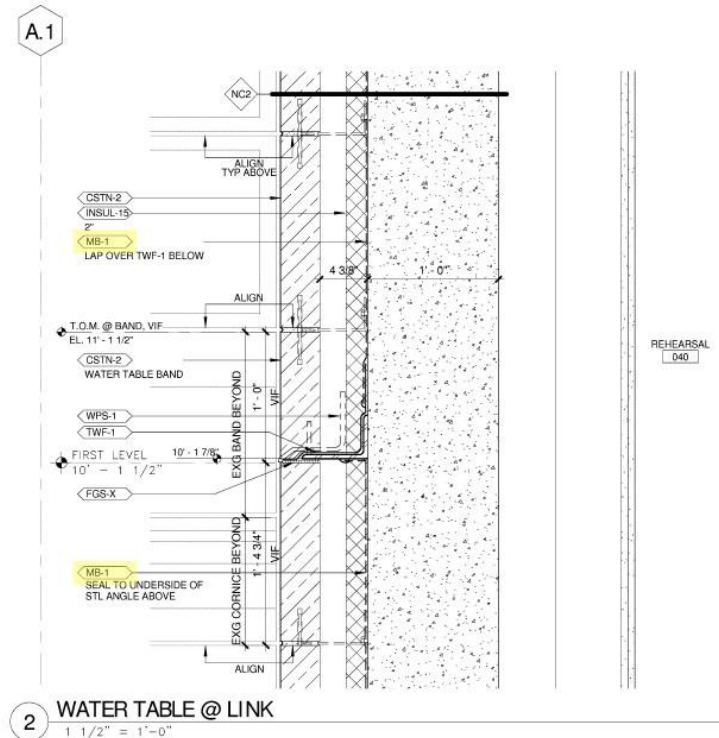


Figure 4 (Source: Architect of Record)

5.15 Buildings with Attached Parking Garages

The Auditorium is not attached to a parking garage, therefore this section does not apply.

5.16 Air Classification and Recirculation

The Auditorium spaces are classified as Class 1 or Class 2 as per Table 6-1 in ASHRAE 62.1-2010. However, most if not all air in the performance spaces, audience chamber, and public spaces is returned to AHU-1, AHU-2, and AHU-3. The performance support spaces, which are served by an active chilled beam system, recirculates Class 1 air before being returned the DOAS AHU-5 for heat recovery.

5.17 ETS Air

The Auditorium is a non-smoking facility and all spaces can be classified as ETS-free spaces. Additionally, all inlets for air circulation are located on the roof of the building eliminating any exposure from smoking locations near to the building. Therefore this section's requirements are not applicable.

ASHRAE 62.1 Section 6: Procedures

6.1 General

The outdoor air is deemed acceptable for purposes of ventilation of the Auditorium. The airflow rates for achieving proper amounts of outdoor air follow the ventilation rate procedure and exhaust rate procedure. Natural ventilation strategies are not used in the building.

6.2 Ventilation Rate Procedure

Breathing zone air flows are calculated by using Equation 6-1 from ASHRAE 62.1-2010 for each space.

$$V_{bz} = R_p \times P_z + R_a \times A_z \quad (6-1)$$

A_z: Occupiable area of the zone [ft²]

P_z: Population of the zone during typical usage [People]

R_a: Outdoor air rate per unit area [cfm/ft²]

R_p: Outdoor air rate per person in the zone [cfm/person]

Values for *R_a* and *R_p*, listed by space use, are located in ASHRAE 62.1-2010 Table 6-1 which is included in Figure 1 of Appendix B for reference. The outdoor air requirement for the zone is then calculated based on the Zone Air Distribution Effectiveness (*E_z*). Values for *E_z* can be found in ASHRAE 62.1-2010 Table 6-2, included below (Figure 5) and are based on how the air is supplied and returned to the zone.

$$V_{oz} = V_{bz}/E_z \quad (6-2)$$

TABLE 6-2 Zone Air Distribution Effectiveness

Air Distribution Configuration	<i>E_z</i>
Ceiling supply of cool air.	1.0
Ceiling supply of warm air and floor return.	1.0
Ceiling supply of warm air 15°F (8°C) or more above space temperature and ceiling return.	0.8
Ceiling supply of warm air less than 15°F (8°C) above space temperature and ceiling return provided that the 150 fpm (0.8 m/s) supply air jet reaches to within 4.5 ft (1.4 m) of floor level. <i>Note</i> : For lower velocity supply air, <i>E_z</i> = 0.8.	1.0
Floor supply of cool air and ceiling return provided that the 150 fpm (0.8 m/s) supply jet reaches 4.5 ft (1.4 m) or more above the floor. <i>Note</i> : Most underfloor air distribution systems comply with this proviso.	1.0
Floor supply of cool air and ceiling return, provided low-velocity displacement ventilation achieves unidirectional flow and thermal stratification.	1.2
Floor supply of warm air and floor return.	1.0
Floor supply of warm air and ceiling return.	0.7
Makeup supply drawn in on the opposite side of the room from the exhaust and/or return.	0.8
Makeup supply drawn in near to the exhaust and/or return location.	0.5

1. "Cool air" is air cooler than space temperature.
2. "Warm air" is air warmer than space temperature.
3. "Ceiling" includes any point above the *breathing zone*.
4. "Floor" includes any point below the *breathing zone*.
5. As an alternative to using the above values, *E_z* may be regarded as equal to air change effectiveness determined in accordance with ANSI/ASHRAE Standard 129¹⁷ for all air distribution configurations except unidirectional flow.

Figure 5 (Source: ASHRAE 62.1-2010)

These outdoor airflows are tabulated in Table 1 in Appendix A. The Auditorium’s AHU-1, AHU-2 and AHU-3 fall under the classification of section 6.2.5 Multiple-zone Recirculating System and therefore additional calculations are required to determine the total outdoor air requirement for the building. To determine the effectiveness of the air handling units to recirculate outdoor air (E_v) the Primary Outdoor Air Fraction (Z_p) must be determined from ASHRAE 62.1-2010 Equation 6-5 as follows:

$$Z_p = V_{oz}/V_{pz} \quad (6-5)$$

V_{oz} : the corrected outdoor air required as determined by Equation 6-2 [cfm]

V_{pz} : the total outdoor and recirculated air to the zone based on the design condition. [cfm]

Looking at all the zones in a given system, the maximum Z_p is used to determine E_v from ASHRAE 62.1-2010 Table 6-3, included below (Figure 6).

TABLE 6-3 System Ventilation Efficiency

Max (Z_p)	E_v
≤0.15	1.0
≤0.25	0.9
≤0.35	0.8
≤0.45	0.7
≤0.55	0.6
>0.55	Use Appendix A

1. “Max (Z_p)” refers to the largest value of Z_p , calculated using Equation 6-5, among all the ventilation zones served by the system.
2. For values of Max (Z_p) between 0.15 and 0.55, the corresponding value of E_v may be determined by interpolating the values in the table.
3. The values of E_v in this table are based on a 0.15 average outdoor air fraction for the system (i.e., the ratio of the uncorrected outdoor air intake (V_{ou}) to the total zone primary airflow for all the zones served by the air handler). For systems with higher values of the average outdoor air fraction, this table may result in unrealistically low values of E_v , and the use of Appendix A may yield more practical results.

Figure 6 (Source: ASHRAE 62.1-2010)

Additionally, since the population of the Auditorium will fluctuate the population diversity is calculated for each AHU to determine an uncorrected outdoor air intake (V_{ou}), which is calculated in accordance with ASHRAE 62.1-2010 Equation 6-6.

$$V_{ou} = D \sum_{all\ zones} (R_p \times P_z) + \sum_{all\ zones} (R_a \times A_z) \quad (6-6)$$

$$\text{where, } D = P_s / \sum_{all\ zones} P_z \quad (6-7)$$

P_s : maximum population served by the system at a given time [People]

$\sum_{all\ zones} P_z$: summation of all the zone populations in the building [People]

Lastly, the total outdoor air for each air handling unit is calculated by following ASHRAE 62.1-2010 Equation 6-8.

$$V_{ot} = V_{ou}/E_v \quad (6-8)$$

The final outdoor air requirement is used to size AHU-1, AHU-2, AHU-3 and a summary of the results is tabulated below in Table 1.

AHU-5 is a 100% outdoor air system and therefore the outdoor air intake requirement is sized under section 6.2.4. The total outdoor air required is equal to the summation of the outdoor air required by each zone, in accordance with ASHRAE 61.1-2010 Equation 6-3

$$V_{ot} = \sum_{all\ zones} V_{oZ} \quad (6-3)$$

The final airflow requirements for AHU-5 are included in Table 1 below. AHU-5 is sized for the outdoor air requirement. Additional heating & cooling is supplied through the chilled beam system and finned-tube radiation.

System Name	Total Supply CFM	Total OA	Max Z Crit	AHU Ventilation Efficiency	AHU OA Required	Exhaust CFM	Total OA/Makeup Air Required	AHU OA %
AHU-1 : Public Spaces	68,325	15,475	0.99	0.8	19,343	3,359	19,343	28%
AHU-2 : Audience Chamber	58,325	11,137	0.23	0.8	13,922	0	13,922	24%
AHU-3 : Performance Spaces	32,260	5,973	0.99	0.8	7,466	0	7,466	23%
AHU-5: Performance Support Spaces	9,680	2,307	0.96	0.8	2,884	2,738	2,884	30%

Table 1 System Calculations

Note: AHU-4 was not used and does not exist in the final construction documentation.

6.5 Exhaust Ventilation

Exhaust airflows were taken into account based on ASHRAE 62.1-2010 Table 6-4, included in Figure 2 Appendix B and calculated to achieve proper ventilation requirements. Exhaust ventilation calculations can be seen in Table 1 of Appendix A.

ASHRAE 62.1 Evaluation Conclusions

As indicated previously, the Auditorium utilizes four outdoor air handling units. AHU-1 is sized to accommodate the public spaces, AHU-2 the audience chamber, AHU-3 the performance spaces, and AHU-5 the performance support spaces. (Note: AHU-4 was not used and does not exist in the final construction documentation.) Complete procedural verification calculations for each zone are tabulated in an Excel spreadsheet and summarized in Table 1 of Appendix A.

The Auditorium is fully compliant with ASHRAE Standard 62.1-2010 and in many aspects far exceeds the standards set forth. The systems and design conditions all abide by the criteria required by Standard 62.1-2010 section 5 to prevent harmful contaminants, resist mold growth, and maintain a healthy system through routine maintenance and cleaning. The design team also followed proper prescriptive methods to calculate required outdoor air flows for ventilation purposes in designing ventilation equipment.

ASHRAE Standard 90.1 Evaluation

This section evaluates at how the Auditorium’s equipment and systems compare to ASHRAE Standard 90.1-2010. Efficiency, energy consumption and building construction are the focus of this standard.

ASHRAE 90.1 Section 5: Building Envelope

The Auditorium’s location in Lemna, Minnesota falls under climate zone 6A. This zone is characterized as cold and moist. This area receives heavy snowfall throughout the winter months. Figure 7 below shows the climate zone breakdown by county. Individual county climate zones are also listed in ASHRAE 90.1-2010.

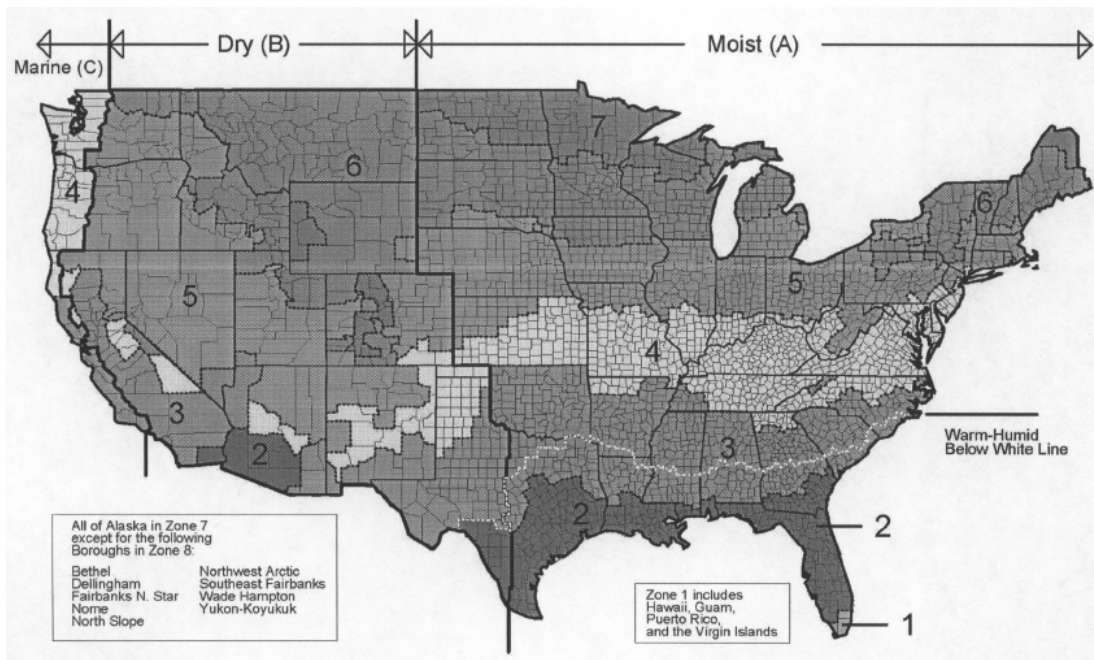


Figure 7 Climate Zones (Source: ASHRAE 90.1-2010)

5.2 Compliance Path

As illustrated in Table 2 below, the window to wall percentage is far below the 40% maximum allowable. Additionally the skylights located on the south roof account for far less than 5% of the building roof area.

	N	S	E	W	Roof
Wall Area	20224	20224	23750	23750	59400
Window/Skylight Area	0	1340	3360	3360	380
% Window Area	0.00%	6.63%	14.15%	14.15%	0.64%

Table 2 Window-Wall Ratio

5.4 Mandatory Provisions

The entrances to the Auditorium each have an enclosed vestibule space in accordance the the standard to separate the exterior environment with the interior conditioned spaces. Also, there is a continuous air barrier enclosing the building. Appropriate insulations and sealants have been utilized to insulated and seal the building against thermal and air leakage. Additionally all fenestration and door are in accordance with the leakage requirements set forth in ASHRAE 90.1-2010 Section 5.4.3.2.

5.5 Prescriptive Building Envelope

Table 3, shown below, details how the above and below grade walls, roof construction, and windows perform against Standard 90.1-2010 Section 5.5 minimum R-values and maximum U-values for climate zone 6A. Considering the building was originally constructed in 1929, the high U-values for the walls, roof and window are acceptable, but noted.

Building Construction						
Type	Description	Actual U-Value		ASHRAE 90.1-2010 U-Value		Compliant [Yes/No]
Wall	Face brick, 12" HW Conc, 1" Insul	0.168	btu/h-ft ² -F	0.080	btu/h-ft ² -F	No
Below-Grade Wall	12" HW Conc 6" Insul	0.045	btu/h-ft ² -F	0.119	btu/h-ft ² -F	Yes
Roof	Steel Sheet. 4" Insul	0.068	btu/h-ft ² -F	0.048	btu/h-ft ² -F	No
Slab	4" LW Concrete	0.213	btu/h-ft ² -F	0.540	btu/h-ft ² -F	Yes
Window	Double Clear 1/4"	0.600	btu/h-ft ² -F	0.550	btu/h-ft ² -F	No

Table 3 - Building Construction Thermal Resistance

ASHRAE 90.1 Section 6: Heating, Ventilation and Air Conditioning

6.4 Mandatory Provisions

All equipment selected in The Auditorium meets the minimum efficiency standards in accordance with ASHRAE 90.1-2010 Table 6.8.1A through Table 6.8.1H and Table 6.8.1K. The mechanical system does not use a variable refrigerant flow system, therefore those standards are not applicable. All load calculations followed prescriptive methods listed in ANSI/ASHRAE/ACCA Standard 183-2007 to size systems and equipment. Additionally, all zones are controlled individually by thermostat controls and all equipment is controlled in accordance with ASHRAE 90.1-2010 Section 6.4.3. Furthermore all equipment construction, installation and insulation follows the standard requirements listed in ASHRAE 90.1-2010 Section 6.4.4.

6.5 Prescriptive Path

The air handling equipment is equipped with economizers and control systems that are in accordance with ASHRAE 90.1-2010 Section 6.5.1. Additionally, the fan power levels comply with the requirements in ASHRAE 90.1-2010 Tables 6.5.3.1.1A/B, included in Figure 3 of Appendix B.

Per Table 1 located in section [6.2 Ventilation Rate Procedure](#) the percentage of outdoor air required at design condition for AHU-1, AHU-2, and AHU-3 are below the required amount to need an energy recovery system per ASHRAE 90.1-2010 Section 6.5.6 Table 6.5.6.1 (Figure 8).

AHU-5 is a dedicated outdoor air handling system and utilizes a dual wheel energy recovery system.

TABLE 6.5.6.1 Energy Recovery Requirement

Zone	% Outdoor Air at Full Design Airflow Rate					
	≥30% and < 40%	≥40% and < 50%	≥50% and < 60%	≥60% and < 70%	≥70% and < 80%	≥80%
	Design Supply Fan Airflow Rate (cfm)					
3B, 3C, 4B, 4C, 5B	NR	NR	NR	NR	≥5000	≥5000
1B, 2B, 5C	NR	NR	≥26000	≥12000	≥5000	≥4000
6B	≥11000	≥5500	≥4500	≥3500	≥2500	≥1500
1A, 2A, 3A, 4A, 5A, 6A	≥5500	≥4500	≥3500	≥2000	≥1000	>0
7, 8	≥2500	≥1000	>0	>0	>0	>0

NR—Not required

Figure 8 (Source: ASHRAE 90.1.-2010)

ASHRAE 90.1 Section 7: Service Water Heating

The service water for heating purposes is supplied through a flood steam heat exchanger, from which the steam supplied comes from the campus steam generation plant at 12,000lbs/hr of steam. All equipment and piping located in the building is properly insulated in accordance with section 7.4.3. Additionally, the control system is controlled from a temperature sensor on the condensate supply side to regulate the amount of heat needed.

ASHRAE 90.1 Section 8: Power

The 2007 Minnesota Building adopted the 2008 National Electrical Code (NEC) which supersedes this section. The 2008 NEC specifies a 3% voltage drop on all feeders and 5% voltage drop on feeders and inclusive branch circuits. The Auditorium power requirements follow these requirements.

ASHRAE 90.1 Section 9: Lighting

The lighting controls for the Auditorium are broken down into different control zones:

- 24hr Egress Lighting,
- Digital Time Control,
- Occupancy Sensor,
- Stand Alone Preset Dimming,
- Time of Day with Maintenance/Security Override
- Time of Day without Override
- Time of Day with User Override

Figure 9, shown on the following page details an example break down of these zones for the second level lighting plan. These controls are in compliance with ASHRAE 90.1 Section 9. The audience chamber and performance lighting are user controlled depending on the performance.

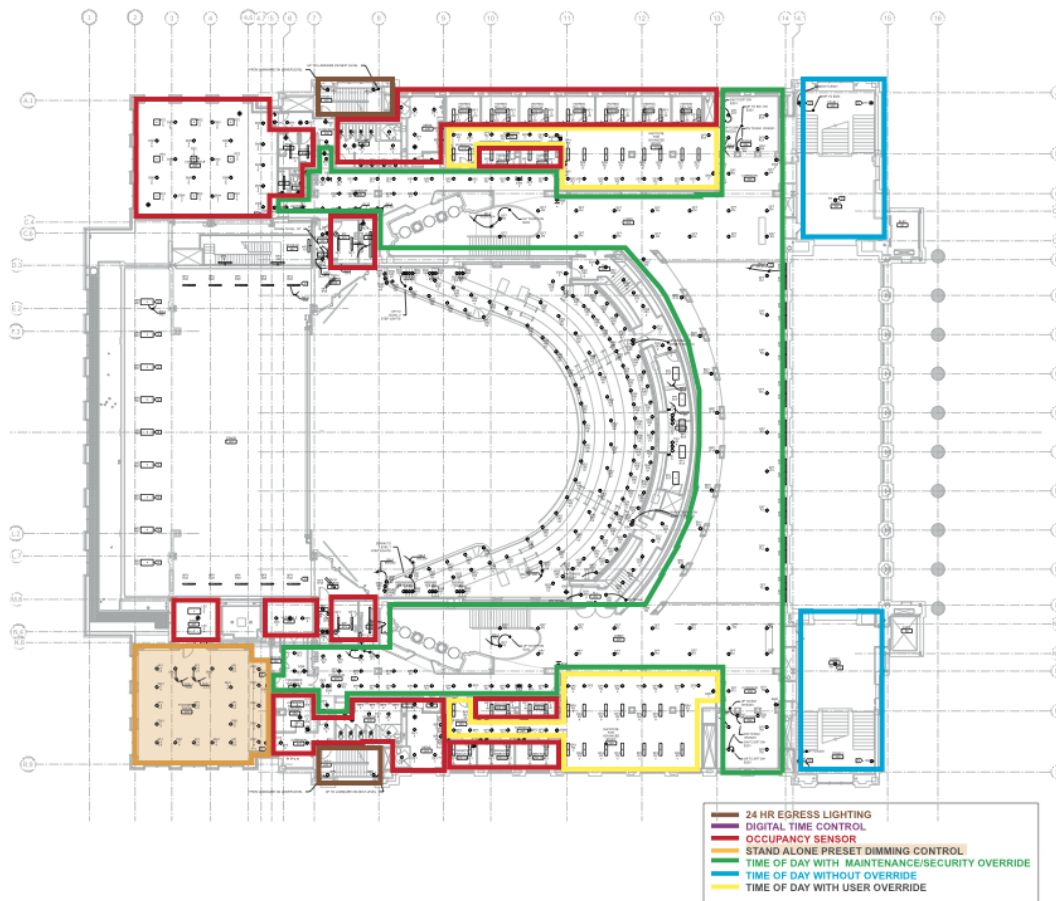


Figure 9 Lighting Control Zones (Source: Architect of Record)

The Space-by-Space method for determining the lighting power densities is tabulated in Table 3 included in Appendix A. All the spaces in the Auditorium comply with the standard limits.

ASHRAE 90.1 Section 10: Other Equipment

Per specification section 230513 Common Motor Requirements for HVAC Equipment, NEMA MG1 is cited as the requirements for all motors in the Auditorium. This meets the requirements set by ASHRAE 90.1-2010 section 10.

ASHRAE 90.1 Evaluation Conclusions

The systems used in the Auditorium are in general compliance with ASHRAE 90.1-2010 considering the historic nature of the building. All of the heating and cooling systems updated or replaced, comply, if not exceed, the standard efficiencies and power requirements set forth. The area where the Auditorium lacks compliance is in the performance of the building envelope, which is to be expected. Improving the quality of insulation and air tightness of the building could cause more thermal and moisture problems for the building later on.

References

ANSI/ASHRAE. (2010). *Standard 62.1-2010, Ventilation for Acceptable Indoor Air Quality*. Atlanta, GA: American Society of Heating Refrigeration and Air Conditioning Engineers, Inc.

ANSI/ASHRAE. (2010). *Standard 90.1-2010, Energy Standard for Buildings Except Low Rise Residential Buildings*. Atlanta, GA: American Society of Heating Refrigeration and Air Conditioning Engineers, Inc.

Note: *At the request of the owner, the identity of the project team is not to be published. For the sources related to the drawings or specifications referenced, please contact Erin Miller at erin.c.miller@psu.edu.*

Appendix A

<i>Table 1 - Ventilation Calculations</i>	19
<i>Table 2 - Lighting Power Densities</i>	23

Room Number and Description	Manual Inputs										Calculations										Engineering Checks										Lookups				
	Area ft ²	Height ft	People	Load Calc Peak cfm	People Density	ASHRAE 62 Zone Distribution System (Pakdown Menu)	Heating Min cfm	Exhaust cfm	Peak cfm	OA cfm	Z Crit	People #	Heating cfm	Exhaust cfm	Peak ACHR	Peak CHMT	OA CHMT	Processed	ASHRAE (chpssion)	ASHRAE CHMT	ASHRAE People Density	Zone Dist. Electronics	People OA cfm	Area OA cfm											
0621 Loading	47	8	3	175	100%	A-Ceiling supply, cooling	30%	0	35	0	0.25	3	53	0	5.4	0.52	0.36	0.14	7.5	0.18	70	1.0	0	0	0										
063 CUST Change	49	8	1	7	100%	A-Ceiling supply, cooling	30%	49	0	0	0.00	1	3	48	1.5	0.20	0.00	0.07	0	0.18	0	1.0	0	0	0										
064 CUST Janitorial	49	8	1	7	100%	A-Ceiling supply, cooling	30%	49	0	0	0.00	1	3	48	1.5	0.20	0.00	0.07	0	0.18	0	1.0	0	0	0										
065 Corridor	2,001	8	2	204	100%	A-Ceiling supply, cooling	30%	0	205	120	0.59	2	62	0	0.8	0.10	0.06	0.04	0	0.06	0	1.0	0	0	120										
066 Restrooms	48	9	1	29	100%	A-Ceiling supply, cooling	30%	24	30	16	0.54	1	9	24	4.2	0.63	0.34	0.22	7.5	0.18	70	1.0	8	9											
069 Corridor	79	8	1	8	100%	A-Ceiling supply, cooling	30%	0	5	5	0.48	1	3	0	0.9	0.13	0.06	0.04	0	0.06	0	1.0	0	5											
070 Dressing Star	139	8	6	94	100%	A-Ceiling supply, cooling	30%	69	35	0	0.00	6	29	69	5.1	0.89	0.00	0.24	0	0	0	1.0	0	0											
071 Dressing Locker Room	157	8	1	157	100%	A-Ceiling supply, cooling	30%	86	160	0	0.00	1	48	76	7.7	1.02	0.00	0.36	0	0.18	70	1.0	0	0											
072 Vesting Comp	157	8	1	157	100%	A-Ceiling supply, cooling	30%	76	160	0	0.00	1	48	76	7.7	1.02	0.00	0.36	0	0.18	70	1.0	0	0											
073 Elec Storage rooms	78	8	1	9	100%	A-Ceiling supply, cooling	30%	0	10	9	0.94	1	3	0	1.0	0.13	0.12	0.04	0	0.12	0	1.0	0	9											
074 Elec Storage rooms	72	8	1	9	100%	A-Ceiling supply, cooling	30%	0	10	9	0.86	1	3	0	1.0	0.14	0.05	0	0.12	0	1.0	0	0	9											
080 Open Office	762	8	5	257	100%	A-Ceiling supply, cooling	30%	0	260	71	0.27	5	78	0	2.6	0.34	0.09	0.12	5	0.06	5	1.0	25	46											
080.2 Mtg Dir	166	8	2	82	100%	A-Ceiling supply, cooling	30%	0	85	20	0.23	2	26	0	3.8	0.51	0.12	0.18	5	0.06	5	1.0	10	10											
081.4 Office space	145	8	2	73	100%	A-Ceiling supply, cooling	30%	0	65	19	0.25	2	23	0	3.9	0.52	0.13	0.18	5	0.06	5	1.0	10	10											
082.4 Office space	102	8	4	31	100%	A-Ceiling supply, cooling	30%	0	65	31	0.30	4	11	0	2.6	0.34	0.10	0.12	5	0.06	5	1.0	5	6											
084 House Mng	279	8	3	84	100%	A-Ceiling supply, cooling	30%	0	85	32	0.37	3	26	0	2.3	0.30	0.11	0.11	5	0.06	5	1.0	15	17											

Lighting Power Density Analysis

SPACE NAME	Area [SF]	LPD [W/SF]	Allowable [Watts]
003 Lobby	572	2.00	1144
003.1 Unisex	58	1.31	77
004 Mens	298	1.31	390
008 First Aid	99	1.00	99
009 Fire Command	152	0.63	96
010 Womens	591	1.31	775
011 Coat Check	169	1.00	169
013 Box office	420	1.00	420
017 Call Center	219	1.00	219
020 Reception	172	2.00	343
020.1 Ticketing Dir	126	1.00	126
020.2 Open Office	631	1.00	631
020.4 Ex Relations Dir	112	1.00	112
020.6 Exec Art Dir	134	1.00	134
029 Unisex	38	1.31	49
030 Green Room	552	2.43	1342
031 Elec	79	0.63	50
036 Corridor	331	0.66	219
040 REHERSAL	1707	2.43	4148
041 Quick Change	39	0.75	30
045 Corridor	2894	0.66	1910
051 STAGE	5655	2.43	13742
055 Crossover	1496	0.63	942
059 STAGE STORAGE	165	0.63	104
060 Loading	1298	0.63	818
061 Tool Stor	131	0.63	83
062 Loading	317	0.63	200
063 Quick Change	42	1.31	55
064 CUST	49	0.95	47
065 Corridor	2001	0.66	1320
066 Unisex	48	1.31	63
069 Corridor	79	0.66	52
070 Dressing Star	139	0.75	104
070.1 TLT	73	1.31	95
072 Visiting Comp	157	0.75	118
074 Corridor	294	0.66	194
077 Elec	78	0.63	49
077.1 Stor	72	0.63	45
080 Open Office	762	1.00	762
080.2 Mktg Dir	166	1.00	166
080.4 Opp Dir/Bus Mngr	145	1.00	145
082 Security	203	1.00	203
084 House Mngr	102	1.00	102
087 Ushers	279	1.00	279
088 West Entry	565	2.00	1130
089 Will Call	288	1.00	288
090 Open Office	439	1.00	439
090.2 Admin Office	112	1.00	112
090.4 IT Support	112	1.00	112
090.6 Bldg Mngr	146	1.00	146

Lighting Power Density Analysis

SPACE NAME	Area [SF]	LPD [W/SF]	Allowable [Watts]
091 Coat Check	448	1.00	448
093 Womens	473	1.31	619
097 Lobby	706	2.00	1412
097 Unisex	61	1.31	80
097.2 Vending	69	0.89	62
098 Mens	330	1.31	432
100 MEM HALL	2878	0.66	1900
101 LOBBY	616	2.00	1232
104 LOUNGE	298	2.00	595
105 CORRIDOR	491	0.66	324
106 CONF	388	1.23	478
107 CONCESSION	255	0.89	227
109 STOR	156	0.63	98
110 INNO LAB	591	1.23	727
110.1 CONF	201	1.23	248
125 CORRIDOR	2032	0.66	1341
128 MENS	220	1.31	288
131 ELEC	74	0.63	47
134 WOMENS	266	1.31	349
137 CORRIDOR	398	0.66	263
150 GROUND LEVEL SEATING	9311	0.79	7356
155 HALL ENTRY	761	2.00	1523
157 LOBBY	4190	2.00	8379
163 VEST	162	2.00	323
168 MENS	233	1.31	305
175 CORRIDOR	2316	0.66	1528
178 WOMENS	265	1.31	348
182 PREP	250	0.89	222
184 SERVING	185	1.24	229
190 COFFEE SHOP	1067	1.24	1323
195 CORRIDOR	503	0.66	332
196 LOUNGE	300	2.00	601
198 LOBBY	614	2.00	1229
201 LOBBY	556	2.00	1111
203 LOBBY	1320	2.00	2639
204 LOUNGE	338	2.00	676
205 CORRIDOR	258	0.66	170
210 INST ADV STD	597	1.23	735
210.02 VST FELLOW	85	1.00	85
210.04 VST FELLOW	109	1.00	109
210.06 VST FELLOW	106	1.00	106
210.08 VST FELLOW	106	1.00	106
210.10 VST FELLOW	106	1.00	106
210.11 VST FELLOW	90	1.00	90
210.12 VST FELLOW	106	1.00	106
210.13 VST FELLOW	86	1.00	86
210.14 VST FELLOW	97	1.00	97
215 CORRIDOR	105	0.66	69
217 WKRM	125	1.23	153
225 CORRIDOR	1303	0.66	860

Lighting Power Density Analysis

SPACE NAME	Area [SF]	LPD [W/SF]	Allowable [Watts]
228 MENS	274	1.31	358
231 ELEC	74	0.63	46
234 WOMENS	266	1.31	348
240 LG CONFERENCE	1528	1.23	1879
241 UNISEX	49	1.31	64
242 CUST	186	0.63	117
244 STOR	61	0.63	38
250 1ST BALCONY	3909	0.79	3088
255 CORRIDOR	962	0.66	635
260 FOUNDERS ROOM	1332	1.23	1638
268 CATERING	145	0.89	129
275 CORRIDOR	1562	0.66	1031
277 ELEC	79	0.63	50
278 WOMENS	266	1.31	349
282 MENS	266	1.31	348
283 WKRM	106	1.23	130
285 CORRIDOR	114	0.66	75
290 INST ADV STD	957	1.23	1177
290.10 DIR	128	1.00	128
290.11 VST FELLOW	90	1.00	90
290.12 MNG DIR	105	1.00	105
290.13 VST FELLOW	95	1.00	95
290.14 GRANTS CNSLT	88	1.00	88
295 CORRIDOR	304	0.66	200
296 LOUNGE	300	2.00	600
297 LOBBY	1296	2.00	2592
298 LOBBY	574	2.00	1148
301 LOBBY	652	2.00	1303
303 LOBBY	1211	2.00	2422
304 LOUNGE	342	2.00	684
305 CORRIDOR	233	0.66	154
307 CORRIDOR	127	0.66	84
310 HONORS	904	1.00	904
310.02 PRO ADVR	85	1.00	85
310.04 PRO ADVR	106	1.00	106
310.06 PRO ADVR	105	1.00	105
310.08 PRO ADVR	106	1.00	106
310.10 PRO ADVR	106	1.00	106
310.11 PRO ADVR	95	1.00	95
310.12 PRO ADVR	106	1.00	106
310.13 PRO ADVR	86	1.00	86
310.14 PRO ADVR	114	1.00	114
310.15 PRO ADVR	90	1.00	90
310.16 PRO ADVR	124	1.00	124
310.17 CORRIDOR	158	0.66	104
318 WKRM	134	1.23	164
327 CORRIDOR	1226	0.66	809
334 WOMENS	233	1.31	305
335 CUST	32	0.63	20
341 MENS	162	1.31	212

Lighting Power Density Analysis

SPACE NAME	Area [SF]	LPD [W/SF]	Allowable [Watts]
350 2ND BALCONY	3909	0.79	3088
356 CORRIDOR	278	0.66	184
368 MENS	166	1.31	218
375 CORRIDOR	1284	0.66	848
377 ELEC	78	0.63	49
378 WOMENS	230	1.31	301
382 CONFERENCE	297	1.23	366
385 CORRIDOR	1527	0.66	1008
390 HONORS	1301	1.00	1301
390.1 ASC DIR	93	1.00	93
390.2 ASC DIR	130	1.00	130
390.3 HNR STD ASSC	89	1.00	89
390.4 HNR DIR	195	1.00	195
390.5 WKRM	121	1.23	149
396 LOUNGE	337	2.00	675
397 LOBBY	1174	2.00	2349
398 LOBBY	619	2.00	1238
400 LEC GALLERY	1483	1.24	1839
403 LOBBY	1389	2.00	2779
404 CORRIDOR	152	0.66	100
416 UNISEX	54	1.31	71
418 MENS	281	1.31	368
420 LOBBY	1230	2.00	2461
422 CONCESSIONS	162	0.89	145
436 PIPE ORGAN LOFT E	619	0.63	390
450 3RD BALCONY	3342	0.79	2640
460 RECITAL HALL	1960	2.43	4762
462 STOR	417	0.63	263
463 CORRIDOR	145	0.66	96
467 CORRIDOR	230	0.66	152
470 ELEC	287	0.63	181
478.1 STOR	54	0.63	34
478.2 STOR	65	0.63	41
482 MISC	86	0.63	54
485 CORRIDOR	1283	0.66	847
485 WOMENS	397	1.31	520
491 VEST	182	0.66	120
493 VEST	249	0.66	165
496 CORRIDOR	93	0.66	62
497 LOBBY	1340	2.00	2679
B08 MECH/STOR	2803	0.63	1766
B08.1 MECH/STOR	1027	0.63	647
B09 FIRE STATION	172	0.63	108
B10 MECH	1748	0.63	1101
B22 CHAIR WAGON SEATING/ORCH PIT	2944	0.79	2326
B23 TLT	228	1.31	299
B27 DRESSING	439	0.75	329
B34 DRESSING	174	0.75	131
B36 TLT	75	1.31	99
B40 DRESSING	175	0.75	131

Lighting Power Density Analysis

SPACE NAME	Area [SF]	LPD [W/SF]	Allowable [Watts]
B42 TLT	76	1.31	100
B43 LKR RM	326	0.75	245
B44 DRESSING	131	0.75	98
B46 TLT	86	1.31	112
B47 UNISEX	44	1.31	58
B48 DRESSING	139	0.75	105
B49 BRK RM	206	0.89	183
B50 TLT	93	1.31	122
B51 TECH DIR	118	1.00	118
B52 DRESSING	131	0.75	98
B53 TECH OFFICE/WKRM	148	1.23	182
B54 TLT	90	1.31	118
B56 DRESSING	214	0.75	161
B57 HOUSE SOUND/ELEC	208	1.00	208
B58 TLT	100	1.31	131
B59 STG MNGR	123	1.00	123
B60 DRESSING	525	0.75	393
B61 WKRM/STOR	173	0.63	109
B62 TLT	241	1.31	316
B63 PROP REPAIR	154	1.00	154
B64 DRESSING	519	0.75	389
B66 TLT	233	1.31	305
B69 MUSICIAN	206	1.00	206
B80 CUST MAIN	202	0.95	192
B82 SOUND WKRM	201	1.00	201
B83 PIANO STOR	200	0.63	126
B84 CORR	3330	0.66	2198
B88 UNISEX	46	1.31	61
B90 LAUNDRY	129	0.60	77
B92 BOH STOR	1128	0.63	711

Appendix B

<i>Figure 1 - ASHRAE 62.1-2010 Table 6-1 - Minimum Ventilation Rates</i>	<i>29</i>
<i>Figure 2 - ASHRAE 62.1-2010 Table 6-4 - Minimum Exhaust Rates</i>	<i>32</i>
<i>Figure 3 - ASHRAE 90.1-2010 Tables 6.5.3.1.1A/B - Fan Power Limitations</i>	<i>33</i>

Appendix B: Figure 1 - ASHRAE 62.1-2010 Table 6-1 - Minimum Ventilation Rates

TABLE 6-1 MINIMUM VENTILATION RATES IN BREATHING ZONE
(This table is not valid in isolation; it must be used in conjunction with the accompanying notes.)

Occupancy Category	People Outdoor Air Rate R_p		Area Outdoor Air Rate R_a		Notes	Default Values			Air Class	
	cfm/person	L/s-person	cfm/ft ²	L/s-m ²		Occupant Density (see Note 4)		Combined Outdoor Air Rate (see Note 5)		
						#/1000 ft ² or #/100 m ²		cfm/person		L/s-person
Correctional Facilities										
Cell	5	2.5	0.12	0.6		25	10	4.9	2	
Dayroom	5	2.5	0.06	0.3		30	7	3.5	1	
Guard stations	5	2.5	0.06	0.3		15	9	4.5	1	
Booking/waiting	7.5	3.8	0.06	0.3		50	9	4.4	2	
Educational Facilities										
Daycare (through age 4)	10	5	0.18	0.9		25	17	8.6	2	
Daycare sickroom	10	5	0.18	0.9		25	17	8.6	3	
Classrooms (ages 5–8)	10	5	0.12	0.6		25	15	7.4	1	
Classrooms (age 9 plus)	10	5	0.12	0.6		35	13	6.7	1	
Lecture classroom	7.5	3.8	0.06	0.3		65	8	4.3	1	
Lecture hall (fixed seats)	7.5	3.8	0.06	0.3		150	8	4.0	1	
Art classroom	10	5	0.18	0.9		20	19	9.5	2	
Science laboratories	10	5	0.18	0.9		25	17	8.6	2	
University/college laboratories	10	5	0.18	0.9		25	17	8.6	2	
Wood/metal shop	10	5	0.18	0.9		20	19	9.5	2	
Computer lab	10	5	0.12	0.6		25	15	7.4	1	
Media center	10	5	0.12	0.6	A	25	15	7.4	1	
Music/theater/dance	10	5	0.06	0.3		35	12	5.9	1	
Multi-use assembly	7.5	3.8	0.06	0.3		100	8	4.1	1	
Food and Beverage Service										
Restaurant dining rooms	7.5	3.8	0.18	0.9		70	10	5.1	2	
Cafeteria/fast-food dining	7.5	3.8	0.18	0.9		100	9	4.7	2	
Bars, cocktail lounges	7.5	3.8	0.18	0.9		100	9	4.7	2	
Kitchen (cooking)	7.5	3.8	0.12	0.6		20	14	7.0	2	
General										
Break rooms	5	2.5	0.06	0.3		25	10	5.1	1	
Coffee stations	5	2.5	0.06	0.3		20	11	5.5	1	
Conference/meeting	5	2.5	0.06	0.3		50	6	3.1	1	
Corridors	–	–	0.06	0.3		–			1	
Occupiable storage rooms for liquids or gels	5	2.5	0.12	0.6	B	2	65	32.5	2	
Hotels, Motels, Resorts, Dormitories										
Bedroom/living room	5	2.5	0.06	0.3		10	11	5.5	1	
Barracks sleeping areas	5	2.5	0.06	0.3		20	8	4.0	1	
Laundry rooms, central	5	2.5	0.12	0.6		10	17	8.5	2	
Laundry rooms within dwelling units	5	2.5	0.12	0.6		10	17	8.5	1	
Lobbies/prefunction	7.5	3.8	0.06	0.3		30	10	4.8	1	
Multipurpose assembly	5	2.5	0.06	0.3		120	6	2.8	1	

Appendix B: Figure 1 - ASHRAE 62.1-2010 Table 6-1 - Minimum Ventilation Rates (con't)

TABLE 6-1 MINIMUM VENTILATION RATES IN BREATHING ZONE (Continued)
(This table is not valid in isolation; it must be used in conjunction with the accompanying notes.)

Occupancy Category	People Outdoor Air Rate R_p		Area Outdoor Air Rate R_a		Notes	Default Values		Air Class	
	cfm/person	L/s-person	cfm/ft ²	L/s-m ²		Occupant Density (see Note 4)	Combined Outdoor Air Rate (see Note 5)		
						#/1000 ft ² or #/100 m ²	cfm/person		L/s-person
Office Buildings									
Breakrooms	5	2.5	0.12	0.6		50	7	3.5	1
Main entry lobbies	5	2.5	0.06	0.3		10	11	5.5	1
Occupiable storage rooms for dry materials	5	2.5	0.06	0.3		2	35	17.5	1
Office space	5	2.5	0.06	0.3		5	17	8.5	1
Reception areas	5	2.5	0.06	0.3		30	7	3.5	1
Telephone/data entry	5	2.5	0.06	0.3		60	6	3.0	1
Miscellaneous Spaces									
Bank vaults/safe deposit	5	2.5	0.06	0.3		5	17	8.5	2
Banks or bank lobbies	7.5	3.8	0.06	0.3		15	12	6.0	1
Computer (not printing)	5	2.5	0.06	0.3		4	20	10.0	1
General manufacturing (excludes heavy industrial and processes using chemicals)	10	5.0	0.18	0.9		7	36	18	3
Pharmacy (prep. area)	5	2.5	0.18	0.9		10	23	11.5	2
Photo studios	5	2.5	0.12	0.6		10	17	8.5	1
Shipping/receiving	10	5	0.12	0.6	B	2	70	35	2
Sorting, packing, light assembly	7.5	3.8	0.12	0.6		7	25	12.5	2
Telephone closets	–	–	0.00	0.0		–			1
Transportation waiting	7.5	3.8	0.06	0.3		100	8	4.1	1
Warehouses	10	5	0.06	0.3	B	–			2
Public Assembly Spaces									
Auditorium seating area	5	2.5	0.06	0.3		150	5	2.7	1
Places of religious worship	5	2.5	0.06	0.3		120	6	2.8	1
Courtrooms	5	2.5	0.06	0.3		70	6	2.9	1
Legislative chambers	5	2.5	0.06	0.3		50	6	3.1	1
Libraries	5	2.5	0.12	0.6		10	17	8.5	1
Lobbies	5	2.5	0.06	0.3		150	5	2.7	1
Museums (children's)	7.5	3.8	0.12	0.6		40	11	5.3	1
Museums/galleries	7.5	3.8	0.06	0.3		40	9	4.6	1
Residential									
Dwelling unit	5	2.5	0.06	0.3	F,G	F			1
Common corridors	–	–	0.06	0.3					1
Retail									
Sales (except as below)	7.5	3.8	0.12	0.6		15	16	7.8	2
Mall common areas	7.5	3.8	0.06	0.3		40	9	4.6	1
Barbershop	7.5	3.8	0.06	0.3		25	10	5.0	2

Appendix B: Figure 1 - ASHRAE 62.1-2010 Table 6-1 - Minimum Ventilation Rates (con't)

TABLE 6-1 MINIMUM VENTILATION RATES IN BREATHING ZONE (Continued)
(This table is not valid in isolation; it must be used in conjunction with the accompanying notes.)

Occupancy Category	People Outdoor Air Rate		Area Outdoor Air Rate		Notes	Default Values		Air Class	
	R_p		R_a			Occupant Density (see Note 4)	Combined Outdoor Air Rate (see Note 5)		
	cfm/person	L/s-person	cfm/ft ²	L/s-m ²			#/1000 ft ² or #/100 m ²		cfm/person
Beauty and nail salons	20	10	0.12	0.6		25	25	12.4	2
Pet shops (animal areas)	7.5	3.8	0.18	0.9		10	26	12.8	2
Supermarket	7.5	3.8	0.06	0.3		8	15	7.6	1
Coin-operated laundries	7.5	3.8	0.12	0.6		20	14	7.0	2
Sports and Entertainment									
Sports arena (play area)	–	–	0.30	1.5	E	–			1
Gym, stadium (play area)	–	–	0.30	1.5		30			2
Spectator areas	7.5	3.8	0.06	0.3		150	8	4.0	1
Swimming (pool & deck)	–	–	0.48	2.4	C	–			2
Disco/dance floors	20	10	0.06	0.3		100	21	10.3	2
Health club/aerobics room	20	10	0.06	0.3		40	22	10.8	2
Health club/weight rooms	20	10	0.06	0.3		10	26	13.0	2
Bowling alley (seating)	10	5	0.12	0.6		40	13	6.5	1
Gambling casinos	7.5	3.8	0.18	0.9		120	9	4.6	1
Game arcades	7.5	3.8	0.18	0.9		20	17	8.3	1
Stages, studios	10	5	0.06	0.3	D	70	11	5.4	1

GENERAL NOTES FOR TABLE 6-1

- 1 **Related requirements:** The rates in this table are based on all other applicable requirements of this standard being met.
- 2 **Environmental Tobacco Smoke:** This table applies to ETS-free areas. Refer to Section 5.17 for requirements for buildings containing ETS areas and ETS-free areas.
- 3 **Air density:** Volumetric airflow rates are based on an air density of 0.075 lb_m/ft³ (1.2 kg_m/m³), which corresponds to dry air at a barometric pressure of 1 atm (101.3 kPa) and an air temperature of 70°F (21°C). Rates may be adjusted for actual density but such adjustment is not required for compliance with this standard.
- 4 **Default occupant density:** The default occupant density shall be used when actual occupant density is not known.
- 5 **Default combined outdoor air rate (per person):** This rate is based on the default occupant density.
- 6 **Unlisted occupancies:** If the occupancy category for a proposed space or zone is not listed, the requirements for the listed occupancy category that is most similar in terms of occupant density, activities and building construction shall be used.

ITEM-SPECIFIC NOTES FOR TABLE 6-1

- A For high school and college libraries, use values shown for Public Assembly Spaces—Libraries.
- B Rate may not be sufficient when stored materials include those having potentially harmful emissions.
- C Rate does not allow for humidity control. Additional ventilation or dehumidification may be required to remove moisture. "Deck area" refers to the area surrounding the pool that would be expected to be wetted during normal pool use, i.e., when the pool is occupied. Deck area that is not expected to be wetted shall be designated as a space type (for example, "spectator area").
- D Rate does not include special exhaust for stage effects, e.g., dry ice vapors, smoke.
- E When combustion equipment is intended to be used on the playing surface, additional dilution ventilation and/or source control shall be provided.
- F Default occupancy for dwelling units shall be two persons for studio and one-bedroom units, with one additional person for each additional bedroom.
- G Air from one residential dwelling shall not be recirculated or transferred to any other space outside of that dwelling.

⋮

Appendix B: Figure 2 - ASHRAE 62.1-2010 Table 6-4 - Minimum Exhaust Rates

TABLE 6-4 Minimum Exhaust Rates

Occupancy Category	Exhaust Rate, cfm/unit	Exhaust Rate, cfm/ft ²	Notes	Exhaust Rate, L/s·unit	Exhaust Rate, L/s·m ²	Air Class
Arenas	–	0.50	B	–	–	1
Art classrooms	–	0.70		–	3.5	2
Auto repair rooms	–	1.50	A	–	7.5	2
Barber shops	–	0.50		–	2.5	2
Beauty and nail salons	–	0.60		–	3.0	2
Cells with toilet	–	1.00		–	5.0	2
Copy, printing rooms	–	0.50		–	2.5	2
Darkrooms	–	1.00		–	5.0	2
Educational science laboratories	–	1.00		–	5.0	2
Janitor closets, trash rooms, recycling	–	1.00		–	5.0	3
Kitchenettes	–	0.30		–	1.5	2
Kitchens—commercial	–	0.70		–	3.5	2
Locker/dressing rooms	–	0.25		–	1.25	2
Locker rooms	–	0.50		–	2.5	2
Paint spray booths	–	–	F	–	–	4
Parking garages	–	0.75	C	–	3.7	2
Pet shops (animal areas)	–	0.90		–	4.5	2
Refrigerating machinery rooms	–	–	F	–	–	3
Residential kitchens	50/100	–	G	25/50	–	2
Soiled laundry storage rooms	–	1.00	F	–	5.0	3
Storage rooms, chemical	–	1.50	F	–	7.5	4
Toilets—private	25/50	–	E	12.5/25	–	2
Toilets—public	50/70	–	D	25/35	–	2
Woodwork shop/classrooms	–	0.50		–	2.5	2

- A Stands where engines are run shall have exhaust systems that directly connect to the engine exhaust and prevent escape of fumes.
 B When combustion equipment is intended to be used on the playing surface additional dilution ventilation and/or source control shall be provided.
 C Exhaust not required if two or more sides comprise walls that are at least 50% open to the outside.
 D Rate is per water closet and/or urinal. Provide the higher rate where periods of heavy use are expected to occur, e.g., toilets in theatres, schools, and sports facilities. The lower rate may be used otherwise.
 E Rate is for a toilet room intended to be occupied by one person at a time. For continuous system operation during normal hours of use, the lower rate may be used. Otherwise use the higher rate.
 F See other applicable standards for exhaust rate.
 G For continuous system operation, the lower rate may be used. Otherwise use the higher rate.

Appendix B: Figure 3 - ASHRAE 90.1-2010 Tables 6.5.3.1.1A/B - Fan Power Limitations

TABLE 6.5.3.1.1A Fan Power Limitation^a

	Limit	Constant Volume	Variable Volume
Option 1: Fan System Motor Nameplate hp	Allowable Nameplate Motor hp	$hp \leq CFM_G \cdot 0.0011$	$hp \leq CFM_G \cdot 0.0015$
Option 2: Fan System bhp	Allowable Fan System bhp	$bhp \leq CFM_G \cdot 0.00094 + A$	$bhp \leq CFM_G \cdot 0.0013 + A$

^awhere
 CFM_G = the maximum design supply airflow rate to conditioned spaces served by the system in cubic feet per minute
 hp = the maximum combined motor nameplate horsepower
 bhp = the maximum combined fan brake horsepower
 A = sum of ($PD \times CFM_G / 4131$)
 where
 PD = each applicable pressure drop adjustment from Table 6.5.3.1.1B in in. w.c.
 CFM_G = the design airflow through each applicable device from Table 6.5.3.1.1B in cubic feet per minute

TABLE 6.5.3.1.1B Fan Power Limitation Pressure Drop Adjustment

Device	Adjustment
Credits	
Fully ducted return and/or exhaust air systems	0.5 in. w.c. (2.15 in. w.c. for laboratory and vivarium systems)
Return and/or exhaust airflow control devices	0.5 in. w.c.
Exhaust filters, scrubbers, or other exhaust treatment	The pressure drop of device calculated at fan system design condition
Particulate Filtration Credit: MERV 9 through 12	0.5 in. w.c.
Particulate Filtration Credit: MERV 13 through 15	0.9 in. w.c.
Particulate Filtration Credit: MERV 16 and greater and electronically enhanced filters	Pressure drop calculated at 2× clean filter pressure drop at fan system design condition
Carbon and other gas-phase air cleaners	Clean filter pressure drop at fan system design condition
Biosafety cabinet	Pressure drop of device at fan system design condition
Energy Recovery Device, other than Coil Runaround Loop	(2.2 × Energy Recovery Effectiveness)—0.5 in w.c. for each airstream
Coil Runaround Loop	0.6 in. w.c. for each airstream
Evaporative humidifier/cooler in series with another cooling coil	Pressure drop of device at fan system design condition
Sound Attenuation Section	0.15 in. w.c.
Exhaust system serving fume hoods	0.35 in. w.c.
Laboratory and vivarium exhaust systems in high-rise buildings	0.25 in. w.c./100 ft of vertical duct exceeding 75 ft